

# Notice No.5

## Rules and Regulations for the Classification of Ships, July 2018

The status of this Rule set is amended as shown and is now to be read in conjunction with this and prior Notices. Any corrigenda included in the Notice are effective immediately.

Please note that corrigenda amends to paragraphs, Tables and Figures are not shown in their entirety.

Issue date: November 2018

Amendments to	Effective date	IACS/IMO implementation (if applicable)
Part 3, Chapter 5, Section 1	Corrigendum	N/A
Part 3, Chapter 13, Sections 2, 7 & 8	Corrigenda	N/A
Part 5, Chapter 2, Sections 1, 3 & 7	Corrigenda	N/A
Part 6, Chapter 2, Section 19	Corrigenda	N/A

## Part 3, Chapter 5 Fore End Structure

### ■ Section 1 General

#### 1.6 Strengthening against bow flare slamming

**Table 5.1.3 Buckling procedure for primary member web plating and web stiffener**

Symbols
$P_W$ = load transmitted through the primary member web stiffener plating, in kN $= P - P_S$ , or by direct calculations

## Part 3, Chapter 13 Ship Control Systems

### ■ Section 2 Rudders

#### 2.7 Rudder torque for rudder blades with cut-outs

2.7.1 The maximum rudder torque,  $Q_R$ , is to be determined from both the ahead and astern conditions as follows:

$$Q_R = C_{Rr} \text{ Nm}$$

$K_4$   $K_1$  = ratio of the rudder blade area forward of the rudder stock centreline, to the rudder blade area:  

$$= \frac{A_f}{A}$$

### ■ Section 7 Equipment

#### 7.1 General

**Table 13.7.1 Equipment requirements**

Ship type	Service	Required equipment
Offshore supply ships	Unrestricted service	(10) As per item (1) using $N_C$ Chain cable length and diameter Increased to correspond to two Equipment and diameter Letters above that required for $N_C$ . Need not be applied for ships with <b>DP(AAA)</b> , <b>DP(AA)</b> , or <b>DP(AM)</b> notations

### ■ Section 8 Anchor windlass design and testing

#### 8.2 Plans and particulars to be submitted

8.2.1 The following plans showing the design specifications, the standard of compliance, engineering analyses and details of construction, as applicable, are to be submitted for evaluation:

- Windlass foundation drawings inclusive of including the supporting structure below deck. The details shall include bolts, chocks, shear stoppers etc., along with the foot print loads for the specified windlass ratings.
- Chain stopper foundation drawings inclusive of including the supporting structure below deck. The details shall include bolts, chocks, shear stoppers etc., along with the foot print loads for the specified rating.

- Windlass arrangement plans showing all the components of the anchoring/mooring system such as the prime mover, shafting, cable lifter, anchors and chain cables; mooring winches, wires and fairleads, if they form part of the windlass machinery; brakes; controls; etc.
- Dimensions, materials, welding details, as applicable, of all torque-transmitting components (shafts, gears, clutches, couplings, coupling bolts, etc.) and all load-bearing components (shaft bearings, cable lifter, sheaves, drums, bed-frames, etc.) components of the windlass and of the winch, where applicable, including brakes, chain stopper (if fitted), and foundation.
- Hydraulic system, to include:
  - (a) piping diagram along with system design pressure;
  - (b) safety valves arrangement and settings;
  - (c) material specifications for pipes and equipment;
  - (d) typical pipe joints, as applicable, and;
  - (e) technical data and details for hydraulic motors;
  - (f) cooling systems arrangements for hydraulic system oil.

### 8.3 Materials and fabrication

8.3.1 Materials used in the construction of torque-transmitting and load-bearing parts of windlasses are to comply with LR's *Rules for the Manufacture, Testing and Certification of Materials, July 2018* or an appropriate National or International Standard acceptable to LR, provided that the Standard gives reasonable equivalence to the requirements of LR. The proposed materials are to be indicated in the construction plans and are to be approved in connection with the design. All such materials are to be certified by the material manufacturers and are to be traceable to the manufacturers' certificates.

### 8.4 Windlass design

8.4.1 In addition to the requirements of the National or International Standard or code of practice acceptable to LR (see *Pt 3, Ch 135, 8.1 General 8.1.2*) the following performance requirements are to be complied with:

- (c) Continuous Duty Pull: The windlass is to have sufficient power to exert a continuous duty pull,  $Z_{cont1}$ , over a period of 30 minutes corresponding to the grade and diameter,  $d_c$ , of the chain cables as follows:

The anchor masses are assumed to be the masses as given in *Table 13.7.2 5.6.1 Equipment — Bower anchors and chain cables Kedge anchors and wires, towlines and mooring lines*. The value of  $Z_{cont}$  is based on the hoisting of one anchor at a time, and also assumes that the effects of buoyancy and hawse pipe efficiency (assumed to be 70 per cent) have been accounted for. In general, stresses in each torque-transmitting component are not to exceed 40 per cent of yield strength (or 0,2 per cent proof stress) of the material under these loading conditions.

8.4.4 The following criteria are to be used for gearing design:

Gears intended to transmit power greater than 100 kW are to be certified by LR, and the gears are to meet the requirements of *Pt 5, Ch 5 Gearing*.

### 8.5 Hydraulic systems

8.5.1 Hydraulic systems, where employed for driving windlasses, are to comply with the requirements of *Pt 5, Ch 14, 9 Hydraulic systems*.

### 8.8 Protection arrangements

8.8.8 The design of the windlass is to be such that the following requirements or equivalent arrangements will minimise the probability of the chain locker or forecastle being flooded in bad weather:

- (a) a weathertight connection can be made between the windlass bedplate, or its equivalent, and the upper end of the chain pipe by means of a cover or seal, and

### 8.9 Shop inspection and testing

8.9.1 Windlasses are to be inspected during fabrication at the manufacturers' facilities by a Surveyor for conformance with the approved plans. Acceptance tests, as specified in the specified Standard (see *Pt 3, Ch 13, 8.1 General 8.1.2*), are to be witnessed by the Surveyor and include the following tests, as a minimum:

- (b) Load test. The windlass is to be tested to verify that the continuous duty pull, overload capacity and hoisting speed as specified in *Pt 3, Ch 13, 8.4 Windlass design 8.4.1* can be achieved.

Where the manufacturing manufacturer's works does not have adequate facilities, these tests, including the adjustment of the overload protection, can be carried out on board ship. In these cases, functional testing in the manufacturer's works is to be performed under no-load conditions.

## Part 5, Chapter 2

# Reciprocating Internal Combustion Engines

### ■ Section 1

## General requirements

#### 1.1 Application

~~4.1.2~~ **1.1.1** The requirements of this Chapter are applicable to reciprocating internal combustion engines operating on liquid, gas or dual fuel for main propulsion and essential auxiliary services (hereinafter referred to as engines). *Pt 5, Ch 2, 3 Crankshaft Design* is not applicable to auxiliary engines having powers of less than 110 kW.

~~4.1.4~~ **1.1.2** Engines providing power for services essential to the safety of the vessel are to be constructed under survey and in accordance with the requirements of this Chapter (see also *Pt 1, Ch 2, 2.4 Class notations (machinery) 2.4.1*).

#### 1.3 Approval process

1.3.4 To apply for an LR Engine Certificate, the following are to be submitted:

- (b) where there is a licensor/licensee arrangement, the list required by *Pt 5, Ch 2, 1.3 Approval process 1.3.4(a)* is to cross-reference the drawings submitted by the designer as part of the engine Type Approval. This list is to identify all changes where the approved design has been modified by the licensee. Where the licensee proposes design modifications to components, a statement is to be made confirming the licensor's acceptance of the proposed changes. If designer/licensor's acceptance is not confirmed, the engine is to be regarded as a different type and is subject to the complete appraisal and type approval process.

### ■ Section 3

## Crankshaft Design

#### 3.2 Scope

3.2.2 This section uses the ~~static~~ **statically** determinate method; alternative methods, including a fully documented stress analysis, will be specially considered.

3.2.6 The design of crankshafts is based on an evaluation of safety against fatigue in the highly stressed areas. The calculation is also based on the assumption that the areas exposed to highest stresses are:

- fillet transitions between the crankpin and web as well as between the journal and web; **and**
- outlets of crankpin oil bores.

3.2.7 When the journal diameter is equal to or larger than the crankpin ~~one~~ **diameter**, the outlets of main journal oil bores are to be formed in a similar way to the crankpin oil bores, otherwise separate documentation of fatigue safety will be specially considered.

3.2.8 Calculation of crankshaft strength consists initially in determining the nominal alternating bending (see *Pt 5, Ch 2, 3.6 Calculation of bending stresses*) and nominal alternating torsional stresses (see *Pt 5, Ch 2, 3.7 Calculation of torsional stresses*) which, ~~when~~ **when** multiplied by the appropriate stress concentration factors (SCF) (see *Pt 5, Ch 2, 3.8 Stress concentration factors*), result in an equivalent alternating stress (uniaxial stress) (see *Pt 5, Ch 2, 3.10 Equivalent alternating stress*). This equivalent alternating stress is then compared with the fatigue strength of the selected crankshaft material (see *Pt 5, Ch 2, 3.11 Fatigue strength*). This comparison will show whether or not the crankshaft concerned is dimensioned adequately (see *Pt 5, Ch 2, 3.12 Acceptability criteria*).

3.2.9 Further information and guidance ~~for on~~ **on** crankshaft design is provided in the LR's ~~Guidance Notes for Crankshaft SCF Calculation using Finite Element Method and Guidance for the Evaluation of Crankshaft Fatigue Tests~~ **Guidance Notes for the Calculation of Stress Concentration Factors, Fatigue Enhancement Methods and Evaluation of Fatigue Tests**.

#### 3.3 Information to be submitted

3.3.1 For the calculation of crankshafts, the documents and particulars listed below are required, ~~this~~ **this**. This information is provided by completing LR Form 2073 and submitting the applicable plans required in *Table 2.1.1 Plans and particulars to be submitted*:

- ~~Crankshaft~~ **crankshaft** drawing (which must contain all data in respect of the geometrical configurations of the crankshaft);
- ~~Type~~ **type** designation and kind of engine (in-line engine or V-type engine with adjacent connecting rods, forked connecting rod or articulated-type connecting rod);
- ~~Operating~~ **operating** and combustion method (2-stroke or 4-stroke cycle/direct injection, precombustion chamber, etc.);
- ~~Number of cylinders;~~
- ~~Output~~ **output** power at maximum continuous rating (MCR), in kW;
- ~~Output~~ **output** speed at maximum continuous power, in rpm;
- ~~Maximum~~ **maximum** firing pressure,  $P_{max}$ , in MPa;

- Mean mean indicated pressure, in MPa;
- Charge charge air pressure (before inlet valves or scavenge ports, whichever applies), in MPa;
- Digitised digitised gas pressure/crank angle cycle for MCR (presented at equidistant intervals at least every 5° CA);
- Mean mean piston speed;
- Compression compression ratio;
- Vee vee angle  $\alpha_v$ , in degrees;
- Firing firing order numbered from driving end, see *Figure 2.3.1 Designation of cylinders*;
- Direction direction of rotation;
- Cylinder cylinder diameter, in mm;
- Piston piston stroke, in mm;
- Centre centre of gravity of connecting rod from large end centre, in mm;
- Radius radius of gyration of connecting rod, in mm;
- Length length of connecting rod between bearing centres,  $L_H$ , in mm;
- Mass mass of single crankweb (indicate if webs either side of pin are of different mass values), in kg;
- Centre centre of gravity of crankweb mass from shaft axis, in mm;
- Mass mass of counterweights fitted (for complete crankshaft) indicate positions fitted, in kg;
- Centre centre of gravity of counterweights (for complete crankshaft) measured from shaft axis, in mm;
- All all individual reciprocating masses acting on one crank, in kg;
- Crankshaft crankshaft material specification(s) (according to ISO, EN, DIN, AISI, etc.);
- Mechanical mechanical properties of material (minimum values obtained from longitudinal test specimens):
  - tensile strength, in N/mm<sup>2</sup>
  - yield strength, in N/mm<sup>2</sup>
  - reduction in area at break, percentage
  - elongation, percentage
- Method method of manufacture (free form forged, continuous grain flow forged, drop-forged, etc., with description of the forging process);
- For for semi-built crankshafts – minimum and maximum diametral interference, in mm; and
- Particulars particulars of alternating torsional stress calculations, see *Pt 5, Ch 2, 3.7 Calculation of torsional stresses*.

3.3.2 The following information is also required for appraisal of the crankshaft (not contained in Form 2073):

- For for engines with articulated-type connecting rod (see *Figure 2.3.2 Articulated-type connecting rod*):
  - Distance distance to link point  $L_A$ , in mm
  - Link link angle  $\alpha_N$ , in degrees
  - Connecting connecting rod length  $L_N$ , in mm
- firing interval (if applicable) i.e. if not evenly distributed;
- Mass mass of connecting rod (including bearings), in kg;
- Mass mass of piston (including piston rod and crosshead where applicable), in kg;
- Every every surface treatment affecting fillets or oil holes shall be specified so as to enable calculation according to Chapter 2 of the LR *Guidance Notes for Crankshaft SCF Calculation using Finite Element Method*;
  - This this is to include Crankshaft fatigue enhancement factors  $K_1$  and  $K_2$  where applicable.
- Maximum maximum alternating torsional stress  $\tau_a$  (N/mm<sup>2</sup>)
- Mechanical mechanical properties of material (minimum values obtained from longitudinal test specimens), in addition to the information listed above:

Impact energy  $K_v$ , in Joules.

### 3.4 Symbols

3.4.1 For the purposes of this Chapter the following symbols apply, see also

- *Figure 2.3.3 Crank dimensions for overlapped crankshaft*
- *Figure 2.3.4 Crank dimensions for crankshaft without overlap*
- *Figure 2.3.5 Crankpin section through the oil bore and*
- *Figure 2.3.6 Crankthrow of semi-built crankshaft.*

$D_{BH}$  = diameter of axial bore in crankpin, in mm

$D_{BG}$  = diameter of axial bore in journal, in mm

$D_G$  = journal diameter, in mm

Note For  $y \geq 0,05D_s$ , where  $y$  is less than  $0,1D_s$ , special consideration is to be given to the effect of the stress due to the shrink fit on the fatigue strength at the crankpin fillet.

$\sigma_{add}$  = additional bending stress due to misalignment and bedplate deformation as well as due to axial and bending vibrations, in N/mm<sup>2</sup>

### 3.6 Calculation of bending stresses

3.6.5 The decisive alternating values will then be calculated according to:

$$X_N = \pm \frac{1}{2}(X_{\max} - X_{\min})$$

where

$X_N$  is considered as the alternating force, moment or stress

$X_{\max}$  is the maximum value within one working cycle

$X_{\min}$  is the minimum value within one working cycle

3.6.9 Alternating bending stresses for the outlet of the crankpin oil bore are calculated as follows:

$$\sigma_{BO} = \pm (\gamma_B \sigma_{BON}) \text{ N/mm}^2$$

### 3.7 Calculation of torsional stresses

3.7.3 Nominal alternating torsional stress is calculated as follows:

$$\tau_N = \pm \frac{M_{TN}}{W_p} \cdot 10^3 \text{ N/mm}^2$$

where

$$M_{TN} = \pm \frac{1}{2}(M_{T\max} - M_{T\min}) \text{ Nm}$$

$$W_p = \frac{\pi}{16} \left( \frac{D^4 - D_{BH}^4}{D} \right) \text{ mm}^3 \text{ for the crankpin, or } W_p = \frac{\pi}{16} \left( \frac{D_G^4 - D_{BG}^4}{D_G} \right) \text{ mm}^3 \text{ for the journal}$$

$\tau_N$  is to be ascertained from assessment of the torsional vibration calculations where the maximum and minimum torques are determined for every mass point of the complete dynamic system and for the entire speed range by means of a harmonic synthesis of the forced vibrations from the first order up to and including the 15th order for 2-stroke cycle engines and from the 0,5th order up to and including the 12th order for 4-stroke cycle engines. Whilst doing so, allowance must be made for the damping that exists in the system and for unfavourable conditions (misfiring in one of the cylinders when no combustion occurs but only on the compression cycle). The speed step calculation shall be selected in such a way that any resonance found in the operational speed range of the engine shall be detected.

### 3.8 Stress concentration factors

**Table 2.3.2 Crankshaft variable boundaries for analytical SCF calculation**

Notes
Low-range The lower bound of $s$ can be extended down to large negative values provided that:
<ul style="list-style-type: none"> <li>• If calculated <math>f(\text{recess}) &lt; 1</math>, then the factor <math>f(\text{recess})</math> is not to be considered (<math>f(\text{recess}) = 1</math>)</li> <li>• If <math>s &lt; -0,5</math>, then <math>f(s, w)</math> and <math>f(r, s)</math> are to be evaluated replacing actual value of <math>s</math> by <math>-0,5</math>.</li> </ul>

### 3.11 Fatigue strength

3.11.1 The fatigue strength is to be understood as that value of equivalent alternating stress (von Mises) which a crankshaft can permanently withstand at the most highly stressed points. The fatigue strength can be evaluated by means of the following formulae:

$$\sigma_{DW} = \pm K(0,42\sigma_B + 39,3) \left[ 0,264 + 1,073D^{-0.2} + \frac{785 - \sigma_B}{4900} + \frac{196}{\sigma_B} \sqrt{\frac{1}{R_x}} \right] \text{ N/mm}^2$$

Where a value of  $K_1$  or  $K_2$  greater than unity is to be applied, then details of the manufacturing process are to be submitted. An enhanced  $K_1$  factor will be considered, subject to special approval of the manufacturing specification. See *Materials and Qualification Procedures for Ships, Book E, Procedure MQPS 5-2*.

3.11.3 Fatigue strength calculations or, alternatively, fatigue test results determined by experiment based either on full size crankthrow (or crankshaft), or on specimens taken from a full size crankthrow, may be required to demonstrate acceptability. The experimental procedure for fatigue evaluation of specimens and fatigue strength of crankshaft assessment are to be submitted for approval by LR. The procedure is to include as a minimum: method, type of specimens, number of specimens (or crankthrows), number of tests, survival probability, and confidence number. See also *LR Guidance for the Evaluation of Crankshaft Fatigue Tests*.

### 3.12 Acceptability criteria

3.12.1 The sufficient dimensioning of a crankshaft is confirmed by a comparison of the equivalent alternating stress and the fatigue strength. The acceptability factor,  $Q$ , is to be greater than or equal to 1,15 for the crankpin fillet, the journal fillet, and the outlet of crankpin oil bore:

$$Q = \frac{\sigma_{DW}}{\sigma_v}$$

### 3.13 Shrink fit of semi-built crankshafts

3.13.4 The maximum permissible internal diameter in the journal pin is to be calculated in accordance with the following formula, this condition serves to avoid plasticity in the hole of the journal pin:

$$D_{BG} = D_S \sqrt{1 - \frac{4000 S_R M_{\max}}{\mu \pi D_S^2 L_S \sigma_{SP}}} \text{ mm}$$

where

$S_R$  = safety factor against slipping; however, a value of not less than 2 is to be taken unless documented by experiments.

$M_{\max}$  = absolute maximum value of the torque  $M_{T\max}$  in accordance with Pt 5, Ch 2, 3.13 Shrink fit of semi-built crankshafts 3.13.7, in Nm

$\mu$  = coefficient for static friction; however, a value of not greater than 0,2 is to be taken unless documented by experiments.

## ■ Section 7 Control and monitoring of main, auxiliary and emergency engines

### 7.3 Auxiliary engine governors

7.3.2 If an engine cannot achieve the requirements of Pt 5, Ch 2, 7.3 Auxiliary engine governors 7.3.1 then the actual load step is to be declared and verified through testing to ensure that the requirements specified in Pt 6, Ch 2, 1.8 Quality of power supplies are satisfied. In cases where a step load equivalent to the rated output of a generator is switched off, a transient speed variation in excess of 10 per cent of the rated speed is acceptable, provided that this does not cause the intervention of the overspeed device as required by Pt 5, Ch 2, 7.4 Overspeed protective devices 7.4.2.

### 7.8 Emergency engines

7.8.1 Alarms and safeguards required are indicated in Table 2.7.4 Emergency engines: Alarms and safeguards.

## Part 6, Chapter 2 Electrical Engineering

## ■ Section 19 Ship safety systems

### 19.1 Watertight doors

19.1.5 The sliding watertight doors on cargo ships are to comply with the following requirements:

- (b) Provisions are to be made as follows:
  - (iii) Availability of the power supply is to be continuously monitored at a point in the electrical circuit adjacent to the door operating equipment. Loss of any such power supply is to activate an audible and visual alarm at the central operating console at on the navigating bridge.
  - (v) The enclosures of electrical components necessarily situated below the bulkhead deck are to provide suitable protection against the ingress of water with ratings as defined in IEC 60529: Degrees of protection provided by enclosures (IP Code), or an acceptable and relevant National Standard, are as follows:
- (vii) An audible alarm, distinct from any other alarm in the area, is to sound whenever the door is closed remotely by power, and sound for at least five seconds but no more than ten seconds before the door begins to move and is to continue sounding until the door is completely closed. The audible alarm is to be supplemented by an intermittent visual signal at the door and in areas where the noise level exceeds 85 dB(A).

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Published by Lloyd's Register Group Limited  
*Registered office* (Reg. no. 08126909)  
71 Fenchurch Street, London, EC3M 4BS  
United Kingdom

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